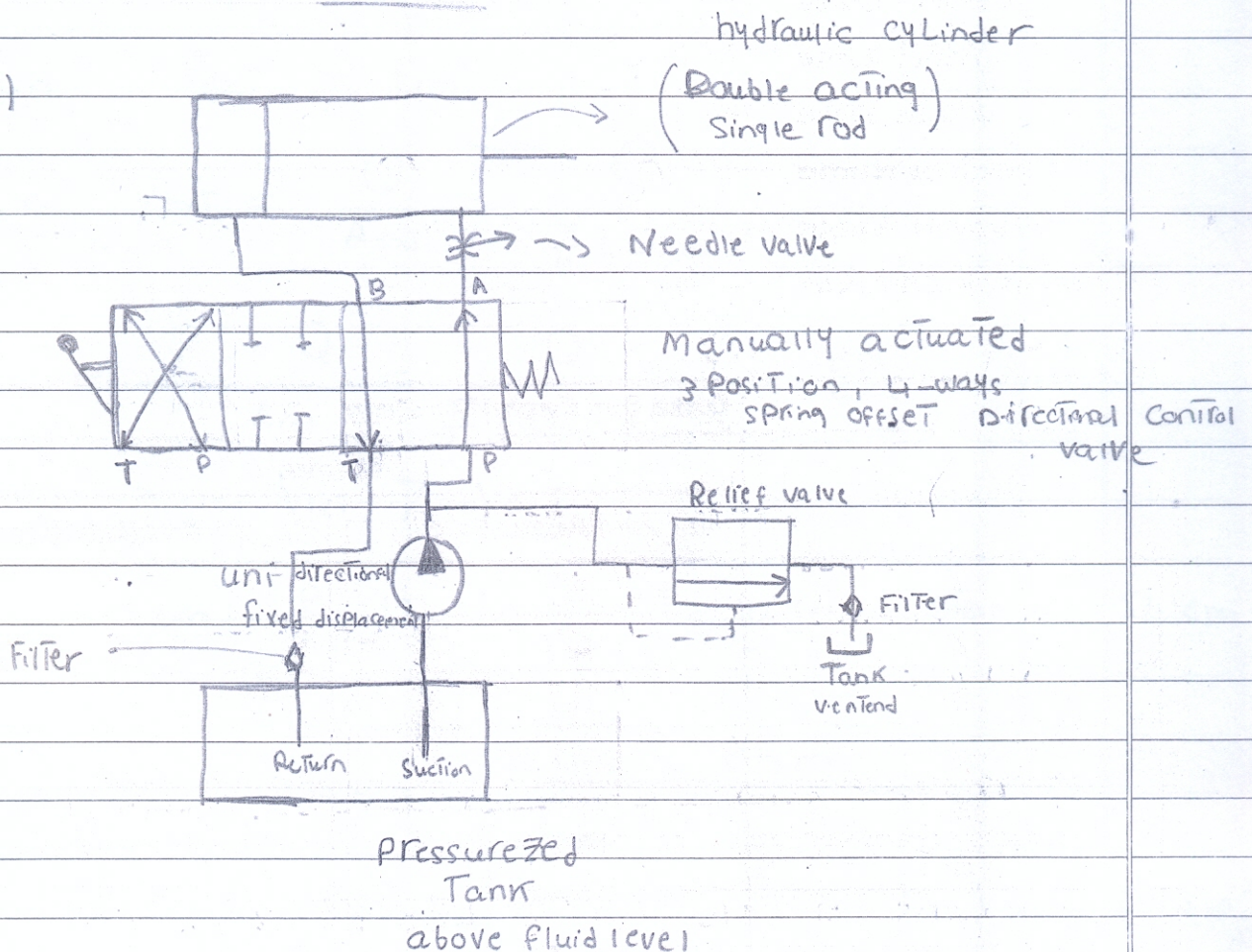


Q.1)

a)

- 1) The cylinder moving Down.
- 2) The valve moved UP in order To reverse The motion of load cylinder.
- 3) The Pressure on The outlet of The Pump Will increase.

4)





b)

$$\therefore F_{EXT} = P A_{EXT}$$

$$F_{EXT} = P \times \frac{\pi}{4} (D_{Piston})^2 \rightarrow \boxed{1}$$

$$\therefore F_{RET} = P \times A_{RET}$$

$$A_{Piston} > A_{Piston} - A_{rod}$$

$$F_{RET} = P \times \frac{\pi}{4} (D_{Piston}^2 - d_{rod}^2) \rightarrow \boxed{2}$$

$$\text{Then } F_{EXT} > F_{RET}$$

$$\therefore V_{EXT} = \frac{Q_i}{A_{EXT}} = \frac{Q}{\frac{\pi}{4} (D_{Piston})^2} \rightarrow \boxed{1}^*$$

$$V_{RET} = \frac{Q}{A_{RET}} = \frac{Q}{\frac{\pi}{4} (D_{Piston}^2 - d_{rod}^2)} \rightarrow \boxed{2}^*$$

$$\text{Then } \boxed{1} \&\boxed{2}^* \quad V_{RET} > V_{EXT}$$

c)  $d = 15 \text{ cm}, P = 124 \text{ bar}, T = 10 \text{ sec}, \text{stroke} = 254 \text{ cm}$   
Power = ??

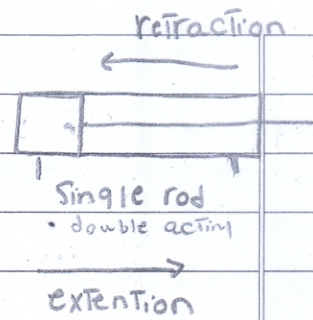
Sol

$$\therefore \text{Power mechanical} = \frac{\text{Force} \times \text{stroke}}{\text{Time}} \rightarrow \boxed{1}$$

$$\therefore \text{Force} = \text{Pressure} \times \text{Area} = (124 \times 10^5) \times \frac{\pi}{4} (15 \times 10^{-2})^2$$

$$= 219126.0876 \text{ N}$$

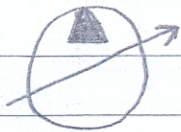
$$\therefore \text{Power} = \frac{219126.0876 \times (254 \times 10^{-2})}{10} = 55658.026 \text{ watt}$$





Q<sub>2</sub>)

a)

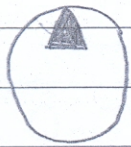


(variable displacement Pump) →

بمعدل كميات متغيرة  
كل دورة cycle

To increase The Pump flow rate in Swash Plate Piston Pump

→ increase Swash Plate angle ( $\theta = \max$ ) →  $V_{D \max}$

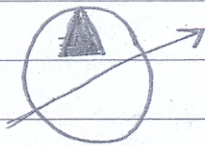


(fixed displacement Pump) →

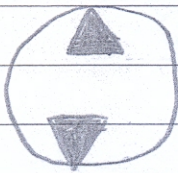
بمعدل كميات ثابتة  
كل دورة cycle (flow rate)

b)

i)



ii)



c)

Positive Pump

Non-Positive Pump

advantage → 1) high Pressure

2) high Volumetric efficiency

3) Small size

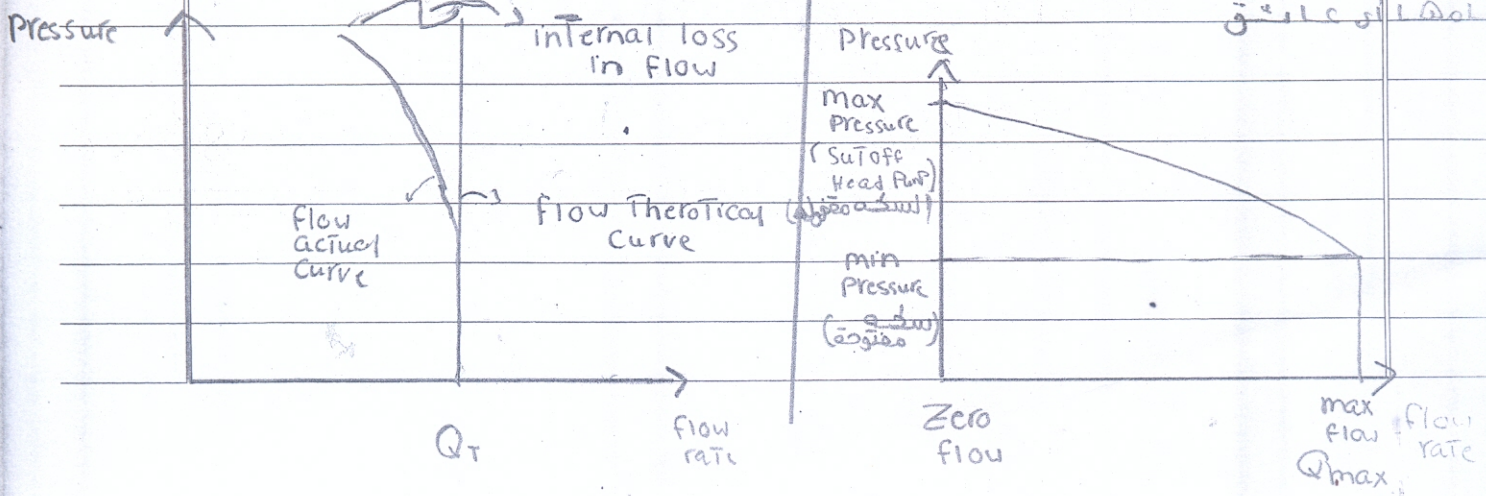
4) small Change in efficiency

5) Great flexibility of Performance

1) Low Pressure

2) Transmission Flow From one location To another

3) easy to install fluid lines  
4) low cost





d) Pressure =  $70 \text{ bar} \times 10^5 \text{ Pa}$

Sol

$$\therefore Q_{act, \text{pump}} = Q_{act, \text{motor}}$$

$$\therefore Q_{Th, \text{pump}} = V_{D, \text{pump}} \times \frac{N}{60} = (82 \times 10^{-6}) \times \frac{500}{60} = 6.83 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\therefore \eta_{vol, \text{pump}} = \frac{Q_{act}}{Q_{Th}}$$

$$\therefore (Q_{act})_{\text{pump}} = 5.6006 \times 10^{-4} \text{ m}^3/\text{sec} = (Q_{act, \text{motor}})$$

$$\eta_{vol, \text{motor}} = \frac{Q_{Th}}{Q_{act}}$$

$$(Q_{Th})_{\text{motor}} = 0.92 \times (5.6006 \times 10^{-4}) = 5.152 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\therefore V_{D, \text{motor}} = \frac{Q_{Th}}{\text{rpm} \times N/60} = \frac{5.152 \times 10^{-4}}{(400/60)} = 7.728 \times 10^{-5} \text{ m}^3$$

$$\begin{aligned} T_{Th, \text{motor}} &= \frac{\text{Pressure} \times Q_{Th}}{2\pi \times 400/60} = \frac{70 \times 10^5 \times (5.152 \times 10^{-4})}{2\pi \times 400/60} \\ &= 86.096 \text{ N.m} \end{aligned}$$

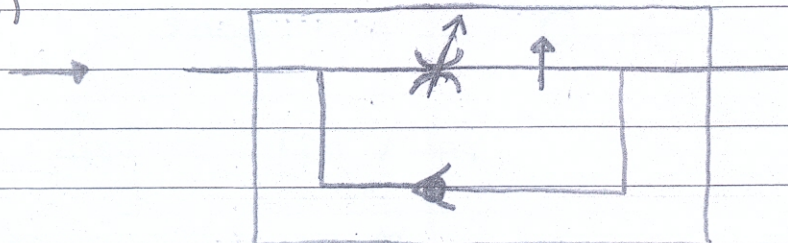
$$\therefore \eta_{mech, \text{motor}} = \frac{T_{act} \times \omega}{\text{Pressure} \times Q_{Th}}$$

$$\therefore T_{act} = 77.48 \text{ N.m}$$

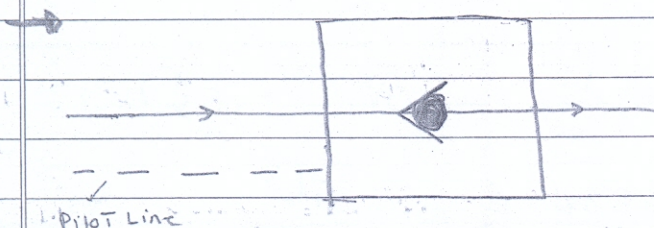


Q3)

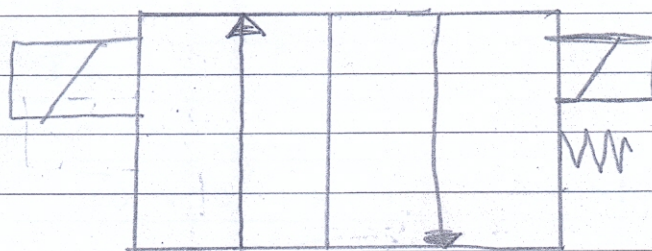
a)



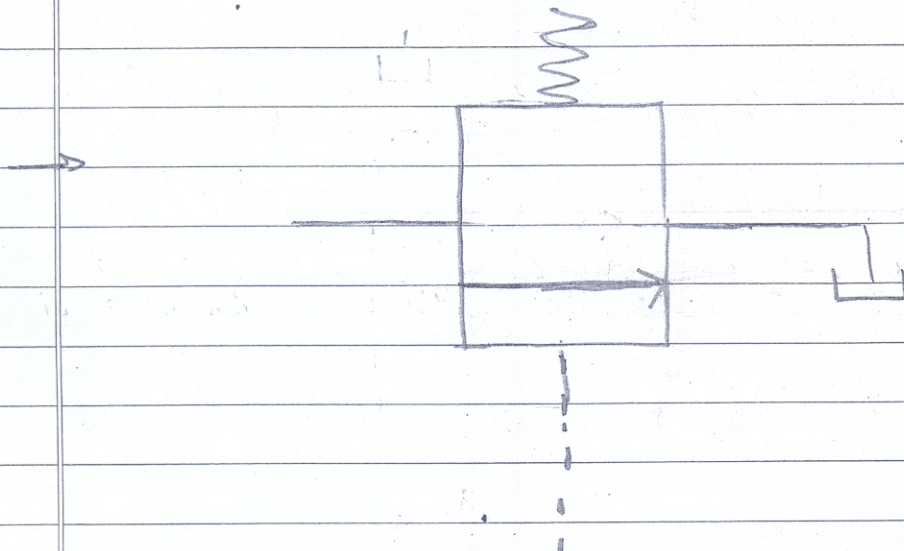
(FCV)  
Pressure compensated



Pilot check valve



DCV



unloading valve



b)  $V_D = 164 \times 10^{-6} \text{ m}^3$ ,  $P = 70 \times 10^5 \text{ Pa}$   
 $N = 2000 \text{ rpm}$ ,  $Q_{act} = 0.006 \text{ m}^3/\text{sec}$

$T_{act} = 170 \text{ N.m}$

find  $\rightarrow \eta_{vol}$ ,  $\eta_{mech}$ ,  $\eta_{overall}$ ,  $P_{act}$  ??

Sol

$\therefore Q_{Th} = V_D \times \frac{N}{60} = 164 \times 10^{-6} \times \frac{2000}{60} = 5.467 \times 10^{-3} \text{ m}^3$

$\therefore \eta_{vol} = \frac{Q_{Th}}{Q_{act}} = \frac{5.467 \times 10^{-3}}{0.006} = 0.911 \times 100 = 91.1\%$

$\therefore \eta_{mech} = \frac{T_{act}}{T_{Th}} \rightarrow$

$\therefore T_{Th} = \frac{70 \times 10^5 \times (5.467 \times 10^{-3})}{2\pi \frac{2000}{60}} = 182.72 \text{ N.m}$

$\therefore \eta_{mech} = \frac{170}{182.72} \times 100 = 0.93 \times 100 = 93.03\%$

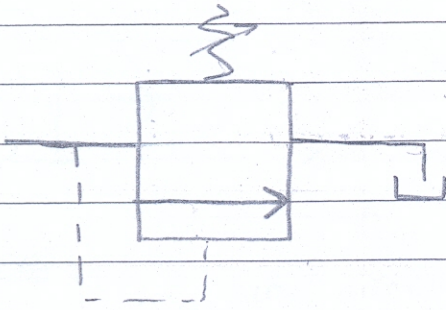
$\eta_{overall} = \eta_{mech} \times \eta_{vol} = 0.847 \times 100 = 84.72\%$

$P_{act_{kcal}} = T_{act} \times \omega = 170 \times \frac{2\pi \times 2000}{60} =$

$= 35604.716 \text{ watt}$



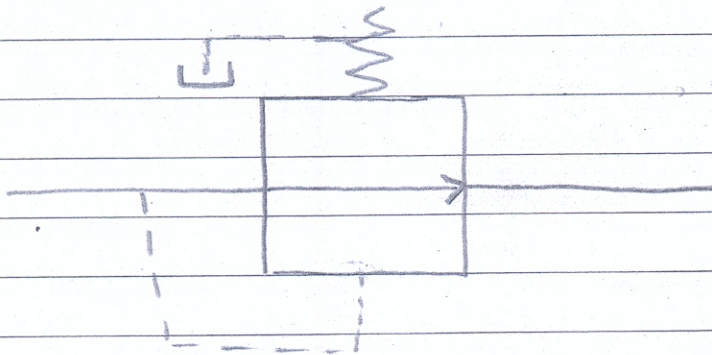
Q4) a)



Relief valve

يتم رفع الضغط إلى الحد الذي Tank  
عندما يزيد الضغط في النظام  
عن الحد المحدد إلى Relief valve

Reducing Valve



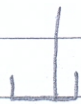
لتكون دائمًا "الضغط" المحدود ويمنع من زيادة الضغط في النظام  
من أن يتجاوز الحد المحدد في النظام ويمنع من ارتفاع الضغط إلى Tank  
والجزء الآخر من النظام



# الجزء الثاني Circuit

b)

i)



Vent line oil Tank  
below fluid level

function

- 1) Storing oil
- 2) Cooling oil
- 3) Separation of air from oil
- 4) Draining of impurities from bottom of Tank



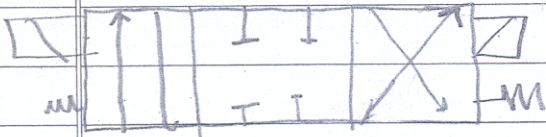
uni directional  
Fixed displacement

function -> To force fluid Through the hydraulic circuit



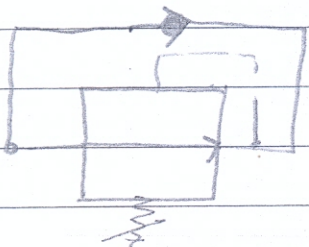
Relief Valve

منه الى الزيت يفرغ الى الخزان Tank الى fluid



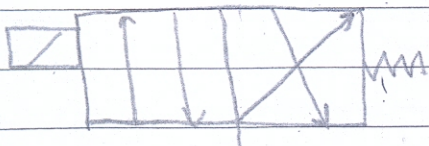
Solenoid actuated 3 Position  
Spring centered, 4-ways

DCV  
التي تفتح في اتجاه واحد flow



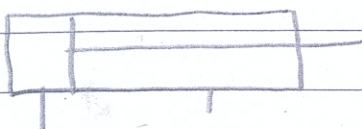
Counter Balance valve

يعمل على منع السائل من النزول الى الاسفل  
من الاسفل الى السطح بسرعة بسبب الوزن والاحتكاك



Solenoid actuated, 2 Position  
Spring offset, 4-ways

DCV  
التي تفتح في اتجاه واحد flow



Double acting, single rod  
hydraulic cylinder

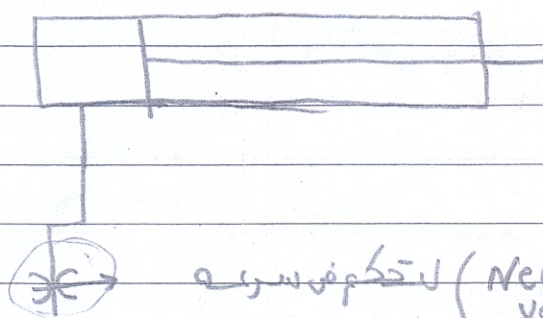
التي وضع extension الى اليمين rod يروح الى اليمين ويتركز الى اليمين  
vertical cylinder الى اليمين

retraction الى اليسار الى اليمين



ii)

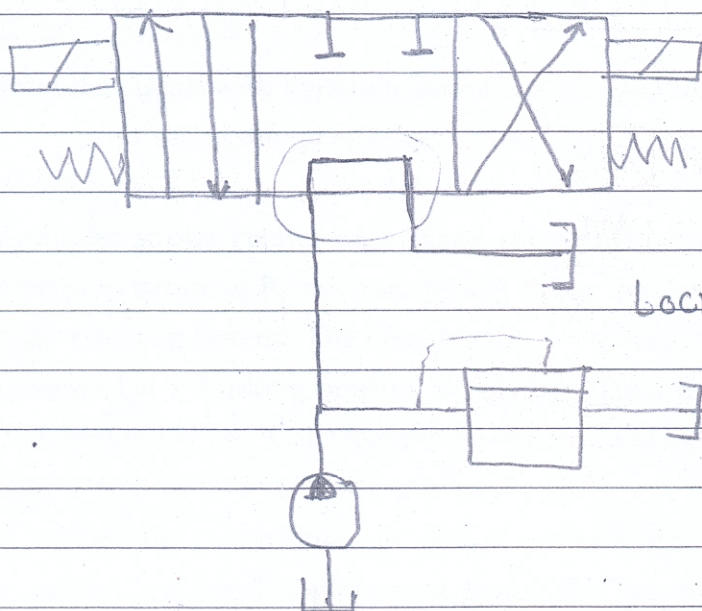
1)



نیڈل وائیو (Needle Valve)

extension side flow

2)



تینج ایل Tank  
فیل و جیج locked

3)

نیڈل وائیو Filter قبل تینج ایل Tank

return line

