

Hydraulic Motors

$$\eta_v = \frac{\text{theoretical flow-rate motor should consume}}{\text{actual flow-rate consumed by motor}} = \frac{Q_T}{Q_A}$$

$$\eta_m = \frac{\text{actual torque delivered by motor}}{\text{torque motor should theoretically deliver}} = \frac{T_A}{T_T}$$

$$T_T (\text{in} \cdot \text{lb}) = \frac{V_D (\text{in}^3/\text{rev}) \times p (\text{psi})}{2\pi}$$

$$T_T (\text{N} \cdot \text{m}) = \frac{V_D (\text{m}^3/\text{rev}) \times p (\text{Pa})}{2\pi}$$

$$\eta_o = \eta_v \eta_m$$

$$\eta_o = \frac{T_A (\text{N} \cdot \text{m}) \times N (\text{rad/s})}{p (\text{Pa}) \times Q_A (\text{m}^3/\text{s})}$$

EXAMPLE 7-5

A hydraulic motor has a 82-cm³ (0.082-L) volumetric displacement. If it has a pressure rating of 70 bars and it receives oil from a 0.0006-m³/s (0.60-Lps or 36.0-Lpm) theoretical flow-rate pump, find the motor

- a. Speed
- b. Theoretical torque
- c. Theoretical power

Solution

- a. From Eq. (7-6M) we solve for the motor speed.

$$N = \frac{Q_T}{V_b} = \frac{0.0006 \text{ m}^3/\text{s}}{0.000082 \text{ m}^3/\text{rev}} = 7.32 \text{ rev/s} = 439 \text{ rpm}$$

- b. Theoretical torque is found using Eq. (7-4M).

$$T_T = \frac{V_D P}{2\pi} = \frac{(0.000082 \text{ m}^3)(70 \times 10^5 \text{ N/m}^2)}{2\pi} = 91.4 \text{ N}\cdot\text{m}$$

- c. Theoretical power is obtained as follows:

$$\begin{aligned} \text{theoretical power} &= T_T N = (91.4 \text{ N}\cdot\text{m})(7.32 \times 2\pi \text{ rad/s}) \\ &= 4200 \text{ W} = 4.20 \text{ kW} \end{aligned}$$

EXAMPLE 7-6

A hydraulic motor has a displacement of 164 cm³ and operates with a pressure of 70 bars and a speed of 2000 rpm. If the actual flow-rate consumed by the motor is 0.006 m³/s and the actual torque delivered by the motor is 170 N · m, find

- a. η_v
- b. η_m
- c. η_o
- d. The actual kW delivered by the motor

Solution

- a. To find the volumetric efficiency, we first calculate the theoretical flow-rate.

$$Q_T = V_b N = (0.000164 \text{ m}^3/\text{rev}) \left(\frac{2000}{60} \text{ rev/s} \right) = 0.00547 \text{ m}^3/\text{s}$$

$$\eta_v = \frac{Q_T}{Q_A} = \frac{0.00547}{0.006} = 0.912 = 91.2\%$$

- b. To find η_m , we need to calculate the theoretical torque.

$$T_T = \frac{V_D P}{2\pi} = \frac{(0.000164)(70 \times 10^5)}{2\pi} = 182.8 \text{ N}\cdot\text{m}$$

$$\eta_m = \frac{T_A}{T_T} = \frac{170}{182.8} = 0.930 = 93.0\%$$

- c. $\eta_o = \eta_v \eta_m = 0.912 \times 0.930 = 0.848 = 84.8\%$

- d. actual power = $T_A N = (170) \left(2000 \times \frac{2\pi}{60} \right) = 35,600 \text{ W} = 35.6 \text{ kW}$