

Q uestion (1) :-

(A) De Fine The Following :-

(1) System :- It is a collection from the component of major engineering disciplines system.

(2) Inputs :- The independent variables of the differential equation.

(3) Response :- refers to external input, initial condition and disturbances.

(B) Give one example (declare input, output and disturbances) for each system :-

(I) Electrical system :-

Example → Television receiver

Input → Frequency of old channel

output → Frequency of New channel

disturbances → Noise

response → Changing channel.

(II) Fluid system :-

Example → City water Tower

Input → water pumped.

output → height of water

disturbances → The amount being used in citizens

response → changing in water level.

### (III) Thermo-mechanical system

Example  $\rightarrow$  Combustion engine

Input  $\rightarrow$  Combustion of fuel & air

Output  $\rightarrow$  movement of engine

Disturbances  $\rightarrow$  Impurities in Gasoline

response  $\rightarrow$  movement of piston

Inside engine

Question (2) :-

(A) Prove the first order differential equation

using free response will be in this form

$$x(t) = x_0 e^{-t/\tau} \quad (\text{use neat sketch}).$$

\* Solution \*

\* Free response

$$u(t) = 0$$

$\tau$  : Time const

$G$  : Gain

$$(\tau D + 1)x = G u(t)$$

$$\tau x' + x = 0 \rightarrow (1)$$

$$x(t) = x_h + x_p$$

$x_h$  : homogeneous soln

$x_p$  : particular soln

$$x_h = A e^{-t/\tau}$$

$$x_p = \text{Zero}$$

$$\left. \begin{aligned} x(t) &= A e^{-t/\tau} \\ x' &= -A/\tau e^{-t/\tau} \end{aligned} \right\} \rightarrow (2)$$



Substituting from equation (2) in equation (1)

$$2\lambda A e^{\lambda t} + A e^{\lambda t} = 0 \quad \div A e^{\lambda t}$$

$$2\lambda + 1 = 0$$

$$\therefore \lambda = -\frac{1}{2}$$

Substituting in equation (2)

$$x(t) = A e^{-t/2} \rightarrow (3)$$

To find  $A$ , apply the initial condition of eqn (3) at time ( $t=0$ )

$$x(0) = x_0 = A e^0$$

$$\therefore A = x_0$$

$$\therefore x(t) = x_0 e^{-t/2}$$

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(B) Determine 1 Time Const ( $\tau$ )

2 response variable

3 external Input

4 Gain ( $G$ )

$$0.006 D z_L + 100 z_L = e_0$$

$$[0.006 D + 100] z_L = e_0 \quad \div 100$$

$$[6 \times 10^{-5} D + 1] z_L = 0.01 e_0.$$

$$(\tau D + 1) x = G u(t)$$

1<sup>st</sup> order differential equation

∴ Time Const =  $6 \times 10^{-5}$  Sec.

response variable  $\rightarrow z_L$

external Input =  $e_0$

Gain = 0.01

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(c) Determine

- 1 response variable
- 2 external input
- 3 natural frequency ( $\omega_n$ )
- 4 damping Ratio ( $\zeta$ )
- 5 Gain ( $G$ )

$$10^{-8} D^2 e_2 + 10^{-5} D e_2 + e_2 = e_0$$

\* solution \*

$$[10^{-8} D^2 + 10^{-5} D + 1] e_2 = e_0$$

2<sup>nd</sup> order differential equation

$$\left[ \frac{1}{\omega_n^2} D^2 + \frac{2\zeta}{\omega_n} D + 1 \right] x = G U(t)$$

$$\rightarrow \frac{1}{\omega_n^2} = 10^{-8} \rightarrow \omega_n = \sqrt{10^{-8}} = 10000 \text{ Hz} \rightarrow \text{Natural Frequency}$$

$$\rightarrow \frac{2\zeta}{\omega_n} = 10^{-5} \rightarrow \zeta = \frac{10^{-5}}{2} \times 10000 = 0.05 < 1$$

(under damped)

↳ damping ratio

→ response variable is  $e_2$

→ Gain ( $G$ ) = 1

→ external Input =  $e_0$

Question (3) :-

Develop, using neat sketch, an equation describing the motion of each of the following

(H)  $m = 5 \text{ Kg.}$

$K = 2000 \text{ N/m}$

$C = 12.5 \text{ N.s/m.}$

Determine the Natural Frequency  $\omega_n = ?$

damping factor  $\zeta = ?$

\* Solution \*

The Free body diagram



$\Sigma \text{ Forces} = m x''$

$m x'' = -Kx - Cx'$

$m x'' + Cx' + Kx = 0$

$[mD^2 + CD + K] x = 0$

$[5D^2 + 12.5D + 2000] x = 0 \quad \div 2000$

$[2.5 \times 10^{-3} D^2 + 6.25 \times 10^{-3} D + 1] x = 0$

2<sup>nd</sup> order differential equation

$\left[ \frac{1}{\omega_n^2} D^2 + \frac{2\zeta}{\omega_n} D + 1 \right] x = G U(t).$



$$\rightarrow \frac{1}{\omega_n^2} = 2.5 \times 10^{-3} \rightarrow \omega_n = \sqrt{\frac{1}{2.5 \times 10^{-3}}} = 20 \text{ Hz}$$

Natural Frequency

$$\rightarrow \frac{2\zeta}{\omega_n} = 6.25 \times 10^{-3}$$

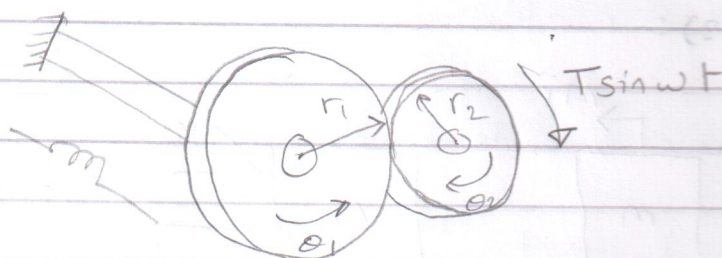
$$\zeta = \frac{6.25 \times 10^{-3}}{2} \times 20 = 0.0625 < 1 \text{ "underdamped"}$$

↳ damping ratio

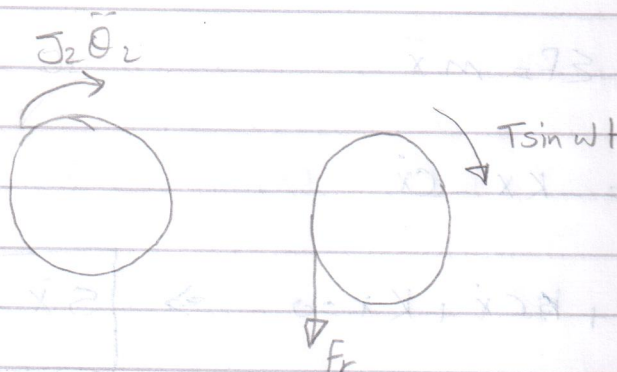
Question (3) → (b)

الف 3

b)



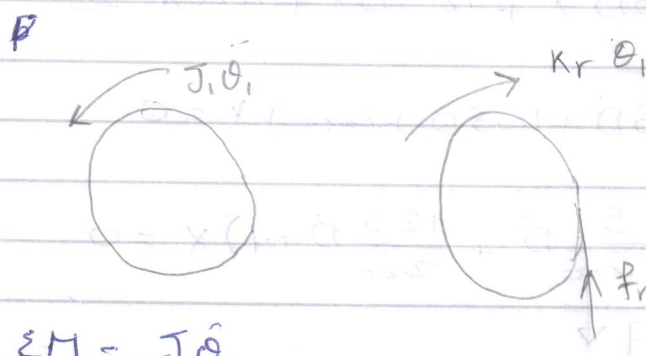
for  $\theta_2$ :



$$\Sigma M = J\ddot{\theta}$$

$$J_2 \ddot{\theta}_2 = T \sin \omega t - F_r \cdot r_2 \Rightarrow (1)$$

for  $\theta_1$ :



$$\Sigma M = J\ddot{\theta}$$

$$J_1 \ddot{\theta}_1 = F_r \cdot r_1 - K_r \theta_1$$

$$F_r = \frac{J_1 \ddot{\theta}_1}{r_1} + \frac{K_r \theta_1}{r_1} = \frac{J_1 \ddot{\theta}_1 + K_r \theta_1}{r_1} \quad (2)$$



Sub From ② in ①

$$J_2 \ddot{\theta}_2 = T \sin \omega t - \left( \frac{J_1 \ddot{\theta}_1 + K_r \theta_1}{r_1} \right) r_2$$

$$J_2 \ddot{\theta}_2 = T \sin \omega t - \frac{J_1 \ddot{\theta}_1 r_2}{r_1} - \frac{K_r r_2 \theta_1}{r_1}$$

$$\therefore J_2 \ddot{\theta}_2 + \frac{J_1 \ddot{\theta}_1 r_2}{r_1} + \frac{K_r r_2 \theta_1}{r_1} = T \sin \omega t$$

$$J_2 \ddot{\theta}_2 + \frac{J_1 \ddot{\theta}_1 (4)}{8} + \frac{K_r (4)}{8} \theta_1 = T \sin \omega t$$